

有侧移对称变截面刚架屈曲临界载荷优化算法

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摘要 针对一类具有侧移的对称变截面刚架屈曲临界载荷求解问题,探讨稳定型下临界载荷的数值优化算法,并提出基于非线性微分方程边值问题的刚架结构临界载荷优化求解算法。**方法** 从立柱起点条件出发,以待求临界荷载、结点弯矩为设计变量,以结点满足连续和边值条件构造目标函数,应用VB语言编制优化程序对算例进行分析计算,并通过ANSYS有限元软件进行仿真对比。**结果** 横梁长为 $2l$,柱高为 l 的等截面刚架的临界载与仿真结果相对误差为1.99%;变截面单梁门式起重机的临界载荷与仿真结果相对误差为2.22%,算法结果与仿真结果相对误差满足精度要求,证实了笔者所提算法的正确性和有效性。**结论** 笔者所提临界载荷优化算法实现了以较少设计变量对临界载荷的高精度计算,为工程中刚架结构稳定性问题的研究提供了参考。

关键词 刚架稳定;非线性微分方程边值问题;优化算法;临界载荷;位型

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The Buckling Critical Load Optimization Algorithm for a Rigid Frame with a Sway Symmetric Variable Cross Section

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Abstract: To solve a class of sway variable cross-section rigid frame with symmetric buckling critical load problem, the numerical optimization algorithms of critical load with the stable buckling position are discussed, and an algorithm of critical load for rigid frame structures based on the boundary values of nonlinear equations is proposed. Starting from the condition of the starting point of the column, the critical load and junction bending moment are considered as design variables, the objective function is established by the conditions of continuity and boundary values of nodes, the numerical examples are analyzed by VB program, and the results are compared by ANSYS

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simulation. The relative error of critical load is 1.99% between the algorithm and simulation of the constant cross-section frame with the beam length being is $2l$ and the column height is l ; The relative error of critical load is 2.22% for variable cross-section single-girder gantry crane, the relative error is satisfied the accuracy requirement, and confirming the correctness and effectiveness of the proposed algorithm. The proposed algorithm achieves high precision calculation of critical load with less design variables, and provides a reference for the study of the stability of rigid frame structures in engineering.

Key words: rigid frame stability; boundary value problem of nonlinear differential equation; optimization algorithm; critical load; buckling position

刚架结构因其自身的特点和性能有着广泛的应用,如机械、土建等工程领域.国内外学者对其静力、动力、屈曲等方面做了大量研究^[1-6].目前对于变截面刚架临界载荷的计算有等效宽度法、有限单元法、能量法等方法. LI Kuinian^[7]考虑了 P-Delta 效应,利用修正因子使刚度矩阵为零,通过一阶分析得到平面框架屈曲载荷. G. N. STAMATOPOULOS 等^[8]采用牛顿-拉斐尔公式,通过迭代计算出刚架屈曲载荷. B. BLOSTOTSKY 等^[9]提出了一种有侧移框架的临界屈曲载荷测量方法.蒋沧如等^[10]考虑初始缺陷影响,基于能量法对刚架结构进行了屈曲分析.干洪等^[11]通过建立刚架稳定方程,用矩阵位移法对平面刚架进行弹性屈曲分析.韩志军等^[12]基于 Hamilton 原理得到了弹性刚架的静力屈曲载荷.

侯祥林等^[13-16]通过将微分方程组离散差分化的方法,提出了计算变截面压杆、刚架结构临界载荷的数值优化算法,此法当节点数量增多时,算法优化难度也会变大,之后又根据边值条件、位型条件等关系计算了变截面压杆临界载荷^[17],在此基础上,笔者通过以临界载荷和未知节点弯矩为设计变量,以非线性屈曲微分方程数值算法按步表达位移与转角为过程,以结构的节点位移和转角满足的边界条件建立目标函数,提出基于非线性微分方程边值问题的刚架结构临界载荷优化求解算法,实现了以较少设计变量对刚架结构的临界载荷的高精度计算,为工程设计

与分析提供参考.

1 屈曲微分方程和边界条件

图 1 为对称变截面刚架受力图,在屈曲临界条件下,位型见图 1(a) 虚线形状.由于对称性,简化结构见图 1(b),当按静力法求解,拆成立柱和横梁两个构件,其受力图如图 1(c) 所示.

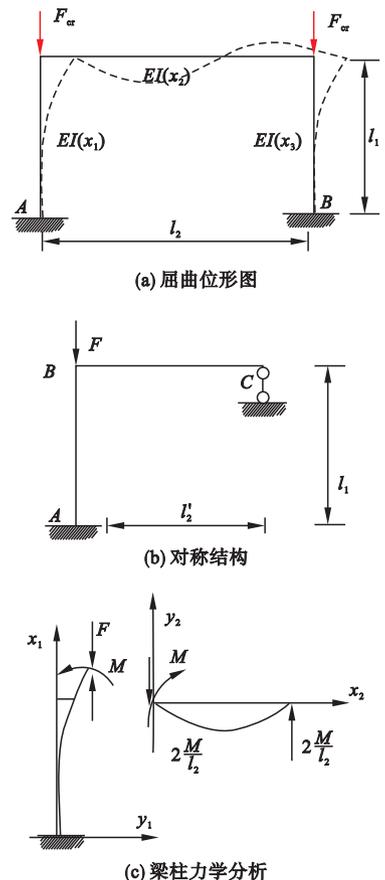


图 1 对称刚架屈曲受力图

Fig. 1 The force diagram of the symmetric frame

由材料力学,可列屈曲位型曲线微分方程为

左右立柱挠曲线的近似平衡方程(忽略剪力):

$$EI_1(x_1)w_1'' = F(\Delta - w_1) - M. \quad (1)$$

梁段近似平衡方程:

$$EI_2(x_2)w_2'' = M - \frac{2M}{l_2}x_2. \quad (2)$$

边界和连续条件:

$$\begin{cases} w_1(0) = 0, w_1'(0) = 0, \\ w_1'(l_1) = w_2'(0), \\ w_2(0) = 0, w_2(l_1') = 0. \end{cases} \quad (3)$$

其中, $EI_1(x_1)$, $EI_2(x_2)$ 为变截面的弯曲刚度.

2 无量纲简化

(1) 柱段

方程式(1)转化为

$$\frac{EI_1(x_1)}{EI_{10}} l_1^2 w_1'' = \frac{F_{cr} l_1^2}{EI_{10}} (\Delta - w_1) - \frac{M l_1^2}{EI_{10}}. \quad (4)$$

为了简化计算,设: $\frac{x_1}{l_1} = t_1$, $\frac{w_1}{l_1} = y_1$, $f_{cr} =$

$$\frac{F_{cr} l_1^2}{EI_{10}}, m = \frac{M l_1}{EI_{10}}, b_1(x_1) = \frac{EI_1(x_1)}{EI_{10}}, \frac{\Delta}{l_1} = \delta, \text{代入}$$

式(4),可得无量纲差分方程:

$$b_1(x_1)y_1'' = f_{cr}(\delta - y_1) - m. \quad (5)$$

设 $Y_1 = y_1$, $Y_2 = y_1'$, 则式(5)转为状态方程:

$$\begin{cases} Y_1' = Y_2, \\ Y_2' = \frac{1}{b_1(x_1)} [f_{cr}(\delta - Y_1) - m]. \end{cases} \quad (6)$$

(2) 梁段

简化过程原理如柱段,式(2)转化为无量纲差分方程式(7),进而转化为状态方程(8).

$$b_2(x_2)y_2'' = \frac{l_2'}{l_1} \frac{EI_{10}}{EI_{20}} m (1 - \frac{x_2}{l_2'}). \quad (7)$$

$$\begin{cases} Y_3' = Y_4, \\ Y_4' = \frac{1}{b_2(x_2)} \frac{l_2'}{l_1} \frac{EI_{10}}{EI_{20}} m (1 - \frac{x_2}{l_2'}). \end{cases} \quad (8)$$

(3) 边界条件

$$\begin{cases} Y_1(0) = 0, \\ Y_2(0) = 0, \\ Y_2(l_1) = Y_4(0). \end{cases} \quad (9)$$

(4) 连续和补充条件

$$\begin{cases} Y_1(l_1) = \delta, \\ Y_3(l_1') = 0. \end{cases} \quad (10)$$

3 临界载荷优化算法原理

3.1 最优化方法

对称变截面刚架临界载荷优化方法构建:

$$\min f(z). \quad (11)$$

设计变量 $z = [z_1, z_2]^T$, z_1 为待求临界载荷, z_2 为节点弯矩, 根据屈曲位型边界条件构建目标函数:

$$f(z) = (Y_1(l_1) - \Delta)^2 + (Y_3(l_1'))^2. \quad (12)$$

3.2 目标函数

(1) 梁柱分段

将长度分别为 $l_i (i = 1, 2)$ 的梁、柱分为 m_i 段, 步长为 $h_i = \frac{l_i}{m_i}$.

(2) 构建目标函数

左右立柱段计算过程. 由初始点 $x_{1,0} = 0$ 的边界条件 $y_1(0) = 0, y_2(0) = 0$ 和式(2), 按 Runge-Kutta 法表达第 $i (i = 1, 2, \dots, m_1)$ 个节点的挠度与转角 $Y_1[z, ih_1], Y_2[z, ih_1]$.

横梁段计算过程. 由 $y_3(0) = 0, y_4(0) = y_2(l_1)$, 按 Runge-Kutta 法表达 $j (j = 1, 2, \dots, m_2)$ 个节点的挠度与转角 $Y_3[z, jh_2], Y_4[z, jh_2]$.

终点边界条件:

$$\begin{cases} Y_1(x_{1,l_1}) = \delta, \\ Y_3(x_{2,l_2}) = 0. \end{cases} \quad (13)$$

归一化目标函数为

$$f(z) = [Y_3(x_{2,l_2})]^2 + [Y_1(x_{1,l_1}) - \delta]^2. \quad (14)$$

3.3 优化程序结构和流程

采用 VB 语言结合优化方法完成程序设计^[18], 包括进退法、黄金分割法、龙格库塔

法、坐标轮换法等程序模块。程序框图如图 2 所示, ξ 为目标函数判别精度。

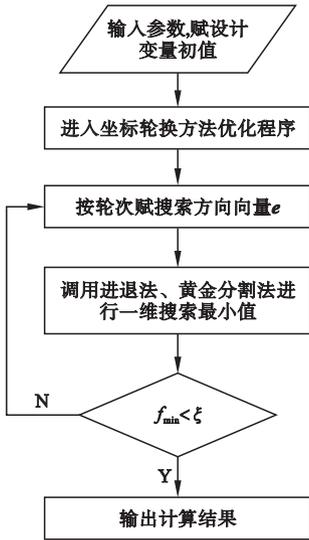


图 2 优化算法程序框图

Fig. 2 Flow chart of the optimization program

4 工程实例分析

(1) 算例 1

图 1 为等截面刚架结构模型, 左右柱脚底端为固定约束, 刚架材料的弹性模量为 E , 惯性矩为 I , 刚架横梁的长度为 $l_2 = 2l$, 左右两边立柱的长度为 $l_1 = l$, 求有侧移对称屈曲临界载荷。

VB 程序优化求解过程。算例中变量 $n = 2$, $z_1 = f_{cr}$, $z_2 = m$, 起点边界条件为: $Y_1(0) = 0$, $Y_2(0) = 0$; 终点边界条件为: $Y_1(l) = 0.1$, $Y_3(l_2) = 0$; 金分割精度为 $e_1 = 10^{-4}$; 多维坐标轮换法精度为 $e = 10^{-11}$; 设计变量初始值 $z_i = \text{Rnd}()$, $i = 1, 2$ 。将计算结果列入表 1, 可以看出经过 6 轮 12 次优化计算, 目标函数达到设定值。

根据程序优化计算结果可得临界载荷和弯矩为

$$F_{cr} = 6.03361 \frac{EI}{l^2}, \quad (15)$$

$$M = 0.26342 \frac{EI}{l}. \quad (16)$$

利用 ANSYS 有限元仿真验证, 模型如

表 1 算例 1 优化过程

Table 1 Optimizaiton proces of example 1

迭代轮数	$z_1 (f_{cr})$	$z_2 (m)$	目标函数值
0	0.70555	0.53342	0.4467830862983
1	7.83294	0.53342	0.0231896188168
1	7.83294	0.24775	0.0014035286991
2	5.86045	0.24775	0.0000984317139
2	5.86045	0.26152	0.0000112218559
3	6.01460	0.26152	0.0000014406750
3	6.01460	0.26326	0.0000001369470
4	6.03214	0.26326	0.0000000105620
4	6.03214	0.26341	0.000000008213
5	6.03348	0.26341	0.000000000568
5	6.03348	0.26342	0.000000000065
6	6.03360	0.26342	0.000000000004
6	6.03361	0.26342	0.000000000001

图 1 所示。左右立柱长度为 $l_1 = 1$ m, 横梁长度为 $l_2 = 2$ m, 截面为边长为 0.1 m 的正方形。采用 BEAM188 梁单元, 单元数为 200, 弹性模量为 $E = 200$ GPa, 泊松比为 $\mu = 0.3$ 。其有限元 ANSYS 仿真计算的临界载荷 $F'_{cr} = 0.986 \times 10^7$ N, 根据式 (15), 计算出笔者算法临界载荷为 $F_{cr} = 10056017$ N。

等截面刚架结构与 ANSYS 仿真计算临界载荷相对误差为

$$\varepsilon = \left| \frac{F_{cr} - F'_{cr}}{F'_{cr}} \right| = 1.99\%.$$

若将误差判别精度设为 3%, 则所述算法满足精度要求, 算法与 ANSYS 仿真屈曲位型曲线对比结果如图 3 所示, 表明笔者算法与有限元软件计算结果非常接近, 验证了笔者算法有效可靠。

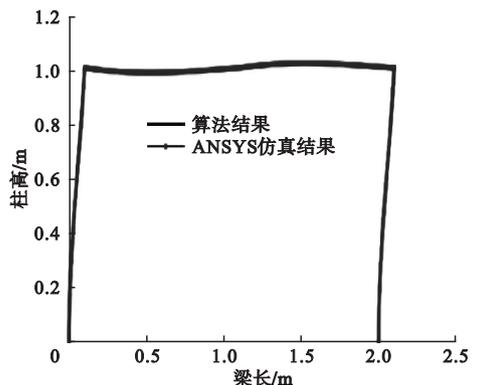


图 3 算例 1 屈曲位形图

Fig. 3 The buckling position of example 1

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